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Primary and Reflected Compaction Waves in a Foam Rod Due to an Axial Impact by a Small Mass

Abstract

The propagation of compaction waves in a stationary foam block subjected to an impact by a small mass is studied in order to examine the mechanism of compaction within the primary and reflected stress waves. The analysis is focused on aluminium strain rate insensitive foam that exhibits strain hardening under quasistatic compression. A theoretical approach is applied using a uniaxial model of compaction in which the compacted strains, being functions of the velocity variation, are not predefined but are obtained as a part of the solution. The present approach allows one to obtain the strain histories and strain distributions within the primary compaction wave as well as within the reflected wave. which propagates in a media with non-uniform density increasing monotonically in the direction of loading. FE simulations considering aluminium based foam Cymat with density 411.5 kg/m3 are carried out in order to verify the proposed theoretical model. A comparison between the impact velocity attenuation predicted by the present model and classical Rigid Perfectly-Plastic Locking material model for cellular materials is discussed.

Keywords

Compaction waves, Reflected wave, Aluminium foam, Impact, Semi-analytical model

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1 INTRODUCTION

The light weight and enhanced energy absorption capacity of cellular materials with different topologies under dynamic loading make these materials attractive for various applications. Therefore efforts are made to develop models of the dynamic compaction when introducing various simplifications. Despite the similar common characteristic of an extended strain region with slowly varying stresses, different mechanisms of compaction develop depending on the material topology. A distinction can be made between cellular materials which exhibit local softening during the response, as honeycombs under out-of-plane impact, and open and closed cellular materials, which exhibit mainly hardening during the quasi-static compression. While the first class of materials is characterized by inertia sensitivity and stress increase proportional to the impact velocity, the latter class of materials is characterized by a dynamic stress enhancement proportional to the velocity squared. Therefore the majority of the theoretical efforts to characterize the cellular materials are focused on the second class.

A shock wave propagation model in cellular materials was proposed by Reid and Peng (1997) to explain the crush enhancement of wood specimens assuming a rigid perfectly-plastic locking (RPPL) mechanism. A thermo-mechanical approach was used in the formulation of the dynamic compaction process to provide a first-order understanding of two impact scenarios (Tan et al., 2005a) supported by the experimental observation of compaction of low density open cell material (Tan et al., 2005b). Retaining the basic characteristics of the one dimensional shock wave models, more detailed material models were used to account for the elastic material properties. An elastic-plastic model with hardening was proposed by Harrigan et al. (2005) while an elastic perfectly-plastic-with rigid locking model was applied by Lopatnikov et al. (2004). Although different material models were assumed, a predefined strain value associated with the fully locked material was used. A summary of different boundary conditions for a uniaxial foam compaction was presented by Main and Gazonas (2008).

While the classical RPPL material model is appropriate and easy to apply for the approximation of the stress-strain characteristic of a cellular material with negligible strain hardening, the majority of higher density cellular materials exhibit strain hardening. The latter becomes significant with the increase of the material density. In order to apply the RPPL model to these materials different definitions of the densification strain have been proposed based on homogeneous material properties (Tan et al., 2005a; Lopatnikov et al., 2004) and based on the cellular topology (Hu and Yu, 2010), all of them being used over the entire range of the velocities, which occur during the impact event.

However, a full compaction within the primary stress wave cannot always be achieved in foam materials exhibiting strain hardening, as shown experimentally by Pattofatto et al. (2007). Tan et al. (2012) also reported experimental results on dynamic compaction of Duocell foam pointing out the dependence of the maximum compaction strains on the impact velocity. In order to analyze the foam compaction under moderate velocity impact, a uniaxial rigid-linear hardening-locking model was proposed by Zheng et al. (2012), which can predict compaction strains that are smaller than the locking strain. A uniaxial shock model, also using a constitutive relation with plastic hardening for cellular materials, was proposed by Zheng et al. (2013) to analyze the compaction under decreasing velocity. The numerical simulations using Voronoi open cell material model and closed-cell foam showed close agreement with the predictions of the analytical model.

Recently an extensive experimental study of the crushing behavior of open-cell Al foam under impact were reported by Barnes et al. (2014) together with a numerical analysis based on the foam microstructural model (Guaitanaros and Kyriakides, 2014). It was revealed that the transition to shock is rather gradual and the analyzed material responds in a quasi-static manner to impact velocities lower than 40 m/s. This findings support the conclusion made in (Karagiozova et al., 2013) that the shock wave theory can be applied only when specific conditions related to the impact velocity and material characteristics are met. Therefore, the concept of a predefined densification strain is not applicable to the analysis of foam materials subjected to a load with a varying velocity or materials with strain hardening. This is particularly true when the applied velocity is a rapidly decreasing function of time, for example the velocity attenuation in a cellular material subjected to a small mass impact (Karagiozova et al., 2012).

Despite the different stress-strain curve approximations, the pre-defined locking strain is the common characteristic of many of the proposed material models. Differently, a model based on a power-law hardening stress-strain characteristic of the foam was used by Pattofatto et al. (2007) to estimate the level of compaction strains due to a constant velocity impact. A more realistic impact scenario when the velocity decreases with time was analyzed in a series of studies by Karagiozova et al. (2010, 2012, 2013, 2014). The proposed approach allowed determining a non-uniform strain distribution behind the wave front with sufficient accuracy. A strong dependence of the strains on the impact velocity within the compacted regions was revealed in these studies.

The commonly used assumption of a locking strain in the theoretical models limits the analysis to the propagation of the primary compaction wave as these models have no capability to deal with a secondary compaction. A simplified model of the propagation of the reflected wave from the layer interface was proposed in (Karagiozova and Alves, 2014) when assuming a nearly constant density caused by the primary wave. However, a secondary compaction due to the reflected wave within a media having a non-uniform foam density, which occurs during the impact by a small mass, has not been examined.

In this paper, the unified approach for dynamic compaction of a class of cellular materials with strain hardening proposed in the previous studies (Karagiozova et al., 2010, 2012, 2013) is extended in order to apply it to the foam compaction within the wave reflected from a stationary boundary of a foam block. The latter compaction is considered as a wave developing in a media with non-uniform initial density which increases monotonically in the direction of loading. Numerical simulations are carried out to verify the results from the theoretical model. An analysis of the mechanism of compaction within the reflected wave can help to estimate the additional portion of the absorbed energy as well as the force transfer at the distal end of a stationary foam block.

2 THEORETICAL MODEL

No details of the cellular topology are considered in the present study and it is assumed that the analyzed class of foam materials can be modeled as a homogeneous material which exhibits strain hardening. The stress-strain dependence for the examined foam is characterized by a strictly concave curve which has a general expression

$$\sigma = g(\varepsilon), \quad g''(\varepsilon) > 0, \quad \varepsilon > \varepsilon_{\gamma} \tag{1}$$

where is the strain at yield. A curve in terms of nominal stress and strain (Langdon et al., 2010) with characteristics defined by Eq. (1) used in the present study is presented in Figure 1. The elastic portion of deformations is neglected when stresses and plastic strains are taken positive in the compression.



Figure 1: Stress-strain characteristic of Cymat with $\rho_0 = 411.5 \text{ kg/m}^3$ (Langdon et al., 2010) together with the equivalent representation of the RPPL model of this material.

2.1 Primary compaction wave

Consider the case when a stationary foam block is subjected to an impact by a small mass with initial velocity. The velocity of the mass is attenuated during the response due to the energy absorbed by the plastic deformation of the foam. In this case, a plastic unloading wave starts to propagate if the applied load causes stresses exceeding the elastic limit of the material. This is a wave of strong discontinuity characterized by discontinuous velocity, stress, strain and density on the wave front while the particles behind the plastic wave front experience elastic unloading (Nowacki, 1978). An elastic precursor wave starts to propagate ahead of the plastic wave front.

Generally, if a plastic unloading wave propagates from region B (behind the wave front) to region A (ahead of the wave front), the conditions through the front of discontinuity, the conservation of mass and momentum conservation, are (Nowacki, 1978)

$$\rho_A(G-V_A) = \rho_B(G-V_B), \quad (\sigma_B - \sigma_A) = -\rho_B(G-V_B)(V_B - V_A).$$
(2a,b)

In Eqs (2), ρ_A, V_A and ρ_B, V_B are the characteristic parameters ahead of and behind the wave front, respectively and is the speed of the wave front.

Since the elastic strains are much smaller than the plastic strains due to compaction, the elastic strains in the deformed foam material behind the front of the unloading wave can be neglected and a rigid unloading can be assumed. The continuity equation in the region of unloading (along the coordinate ξ) is formulated as

$$d\rho_B/dt = -\rho_B \, dV/d\xi \tag{3}$$

Latin American Journal of Solids and Structures 12 (2015) 905-924

which leads to $dV/d\xi = 0$ behind the wave front thus defining a rigid body motion of the compacted region. The particle velocity, V, is independent of ξ and equal to the particle velocity of the wave front.

Schematic of the propagation of the primary and reflected wave in a finite length foam rod when impacted by a mass is shown in Figure 2.



Figure 2: Schematic of the propagation of primary and reflected wave in a finite thickness foam block.

The characteristic variables just behind the primary wave front are

$$V_B = V, \quad \varepsilon_B = \varepsilon_1, \quad \sigma_B = \sigma^d(V), \quad \rho_B = \rho_0/(1 - \varepsilon_B)$$
 (4a-d)

while just ahead of the wave front

$$V_A = 0, \qquad \varepsilon_A = 0, \qquad \sigma_A = \sigma^{qs} = \sigma_Y, \qquad \rho_A = \rho_0$$
 (5a-d)

when neglecting the material elasticity.

The equations of motion, which describe the propagation of the primary compaction wave during the time interval $0 < t \le t_1$, are (Karagiozova et al., 2012)

$$\frac{dV}{dt} = -\frac{1}{m + \rho_0 [(u_1 + h_1)]} \left\{ \sigma_Y + \rho_0 \frac{V^2}{\varepsilon_1(V)} \right\}$$
(6a)

Latin American Journal of Solids and Structures 12 (2015) 905-924

$$\frac{dh_1}{dt} = V \frac{1 - \varepsilon_1}{\varepsilon_1}, \quad G_1(\varepsilon_1) = V/[\varepsilon_1], \qquad \frac{du_1}{dt} = V(t)$$
^(6b-d)

where $V(t), u_1(t)$ are the velocity and displacement of the rigid mass, $h_1(t)$ is the thickness of the compacted zone and m is the mass per unit area of the striker; $\varepsilon_1(V)$ is the compaction strain and $G_1(\varepsilon_1)$ is the Lagrangian speed of the primary compaction wave (Figure 2). The initial conditions are

$$V(0) = V_0, \ h_1(0) = u_1(0) = 0 \tag{7}$$

and $u(t) = u_1(t)$ for $0 \le t \le t_1$. The strain behind the wave front is non-uniform and taking into account the mass conservation law within the compacted region $h_1(t)$, a relationship between the initial density of the foam, ρ_0 and density $\rho(\xi)$ in this region can be obtained

$$\int_{0}^{h_{1}(t)} \rho(\xi) d\xi = \rho_{0} \{ h_{1}(t) + u_{1}(t) \}$$
(8)

Making use of the unique stress-strain relationship of the analyzed foam, a relationship $[\varepsilon_1] = F_{1e}([V])$ can be obtained based on the possibility that the pairs of variables that provide equivalent representation of the Hugoniot curve can be transformed to any of the other possible representations using the jump conditions (Davison, 2008)

$$[V] = ([\sigma][\varepsilon_1]/\rho_0)^{1/2}$$
(9)

Details of the calculation of the function $F_{le}([V])$ are presented in Karagiozova et al. (2012). For the particular foam material, the relationship $[\mathcal{E}_1] = F_{le}([V])$ is best approximated by a function with the following general expression

$$[\varepsilon_1] = F_{1e}([V]) = a_1 \exp(b_1/(V+c_1)) + d_1$$
(10)

and coefficients $a_1 = 116.888449$, $b_1 = -0.607876 \text{m/s}$, $c_1 = 72.314379 \text{m/s}$ and $d_1 = -115.921474$.

2.2 Reflected Compaction Wave

The impact of a small mass on a foam rod causes non-uniform strains behind the primary compaction wave, which decrease rapidly with the distance propagated by the wave during the time interval $0 < t \le t_1$. A reflected wave can occur from the stationary end at $t = t_1$ and develop for $t > t_1$. Since a rigid unloading was assumed behind the primary compaction wave, it can be anticipated that at $t = t_1$, the entire foam rod with length $L_1 = h_1(t_1)$ has velocity $V(t_1)$.

The secondary compaction will develop in a foam rod, which is characterized by a non-uniform initial density $\rho_1(\bar{x})$, ($\bar{x} = L_1 - x$) monotonically increasing in the direction of loading (positive

density gradient). Examples of the material characteristic $\rho_1(\bar{x})$ are shown in Figure 3a for the analyzed Cymat foam when subjected to impacts with initial velocity of 100 and 200 m/s, m = 18.997 kg/m². The initial material strength at $t = t_1$ is assumed to correspond to the attained density $\rho_1(\bar{x})$ so that it can be defined as $\sigma_{1Y}(\bar{x}) = \sigma(\rho_1(\bar{x}), t_1)$, $\rho_1(\bar{x}) = \rho_0/(1 - \varepsilon_1(\bar{x}, t_1))$. The strains $\varepsilon_2(\bar{x})$ along the foam thickness at $t = t_1$, when the second deformation phase commences, are equal to zero due to the assumption of rigid unloading during the propagation of the primary wave.



Figure 3 (a): Material characteristics at the time of initiation of the reflected wave $(t = t_1)$; (a) density distributions for $V_0 = 100$ and 200 m/s.



Figure 3 (b): Material characteristics at the time of initiation of the reflected wave $(t = t_1)$; (b) Stress variation at several cross-sections of the foam, $0 < \overline{x} < L_1$ for $V_0 = 200$ m/s.

The values of $\rho_1(\bar{x})$ and $\sigma_{1Y}(\bar{x})$ at $t = t_1$ are used as characteristics of a material with nonuniform density during the second deformation phase. Since the functions $\rho_1(\bar{x})$ and $\sigma_{1Y}(\bar{x})$ vary monotonically with \bar{x} , they can be approximated by continuous analytical functions. The stress-strain relationship at each cross-section of the foam can be obtained from the original $\sigma(\varepsilon)$ curve when shifting the origin of the curve at $\varepsilon_2(\bar{x}) = \varepsilon(\bar{x}, t_1) - \varepsilon_1(\bar{x}, t_1)$. The stressstrain curves $\sigma_{1Y}(\bar{x}^*, \varepsilon_2)$ at several cross-sections $\bar{x}^* = 0, \bar{x}_1, \bar{x}_2, L_1$ ($0 < \bar{x}_1 < \bar{x}_2 < L_1$) are shown in Figure 3b for a 200 m/s impact.

A rigid unloading behind the reflected wave is assumed similarly to the propagation of the primary wave. The characteristic variables just behind the front of the reflected wave are

$$V_B = 0, \quad \varepsilon_B = \varepsilon_2(\bar{x}), \quad \sigma_B = \sigma_1^d(V), \quad \rho_B = \rho_1(\bar{x})/(1 - \varepsilon_2(\bar{x}))$$
(11a-d)

while just ahead of the wave front

$$V_A = V, \qquad \varepsilon_A = \varepsilon_1(\bar{x}, t_1), \qquad \sigma_A = \sigma_1^{qs} = \sigma_{1Y}(\bar{x}), \qquad \rho_A = \rho_1(\bar{x}) \tag{12a-d}$$

when neglecting the material elasticity. In Eq. (11c),

$$\sigma_1^d(V) = \sigma_{1Y}(\bar{x}) + \rho_1(\bar{x})V^2 / \varepsilon_2(\bar{x}, V)$$
⁽¹³⁾

The equations of motion with respect to the velocity of the rigid mass V(t) and displacement $u_2(t)$ during $t > t_1$ can be obtained from the momentum conservation

$$-\sigma_1^d (\bar{x}, \varepsilon_2(V)) dt = d((m + L\rho_0 - m_2(l_2))V), \quad l_2(t) = u_2(t) + h_2(t)$$
(14)

where $\sigma_1^d(\bar{x}, \varepsilon_2(V))$ is the dynamics stress behind the wave front and $m_2(l_2)$ is the mass per unit area of the new deformed portion near to the stationary end.

Let us assume that the density along the foam rod, $\rho_1(\bar{x})$ can be approximated by the following polynomial function

$$\rho_1(\overline{x}) = \rho_1(\overline{x}, t_1) = \sum_{k=0}^n a_{r,i} (\overline{x}/L_1)^k, \quad \overline{x} \in (0, L_1), \quad n = 4$$
⁽¹⁵⁾

where coefficients a_{ri} have a dimension of density. Then

$$m_{2}(l_{2}) = l_{2} \left(a_{r,0} + \frac{a_{r,1}}{2} \left(\frac{l_{2}}{L_{1}} \right) + \frac{a_{r,2}}{3} \left(\frac{l_{2}}{L_{1}} \right)^{2} + \frac{a_{r,3}}{4} \left(\frac{l_{2}}{L_{1}} \right)^{3} + \frac{a_{r,4}}{5} \left(\frac{l_{2}}{L_{1}} \right)^{4} \right)$$
(16)

The new deformed part has length $l_2(t) = u_2(t) + h_2(t)$ measured from the stationary end, $u_2(t)$ and $h_2(t)$ being the displacement of the moving end and the thickness of the second compacted zone, which develops from the stationary end (Figure 2). Equation (14) can be presented in the form

Latin American Journal of Solids and Structures 12 (2015) 905-924

$$-\sigma_{1}^{d}(\bar{x},\varepsilon_{2}(V)) = (m + L\rho_{0} - m_{2}(l_{2}))\dot{V} - \dot{m}_{2}(l_{2})V, \quad \dot{l}_{2} = \frac{V}{\varepsilon_{2}(\bar{x},V)} \quad .$$
(17a,b)

Substituting Eqs. (13), (16) and (17b) into Eq. (17a), the following equations of motion with respect to V(t), $h_2(t)$ and $u_2(t)$ are obtained

$$\frac{dV}{dt} = -\frac{\sigma_{1Y}(\bar{x})}{m + \left(L\rho_0 - m_2(l_2)\right)},\tag{18a}$$

$$\frac{dh_2}{dt} = V \frac{1 - \varepsilon_2(l_2)}{\varepsilon_2(l_2)}, \quad G_2(l_2) = V \frac{1 - \varepsilon_2(l_2)}{\varepsilon_2(l_2)} \quad , \quad \frac{du_2}{dt} = V(t)$$
(18b,d)

where $\varepsilon_2(l_2) = \varepsilon_2(l_2, V)$ is the compaction strain within the reflected wave and $G_2(l_2)$ (Figure 2) is the Eulerian wave speed of the compaction wave. The initial conditions for the second deformation phases which commences at $t = t_1$ are

$$V(0) = V(t_1), \ h_2(0) = u_2(0) = 0.$$
 (19a,b)

The total displacement of the rigid mass during this deformation phase is

$$u(t) = u_1(t_1) + u_2(t).$$
⁽²⁰⁾

Note that strains $\mathcal{E}_2(\bar{x})$ obtained during the second deformation phase are added to the strains obtained during Phase 1 and the total final strains are obtained as

$$\varepsilon(\bar{x}) = \varepsilon_1(\bar{x}) + \varepsilon_2(\bar{x}). \tag{21}$$

Similarly to the propagation of the primary compaction wave, a relation between the strain jump and the velocity jump can be obtained, which is characteristic at each material point \bar{x} . Therefore the strain jump can be expresses as $[\mathcal{E}_2(\bar{x})] = F_{2e}([V], \bar{x})$. The particular form of this function depends on the foam properties and the stress state attained at the end of the first deformation phase $(t = t_1)$. The governing nonlinear ordinary differential equations (6) and (18) are solved numerically when using Maple 17.

3 NUMERICAL MODEL

Numerical simulations are carried out to verify the one-dimensional theoretical model of foam compaction when analyzing the deformation of a block with diameter 0.05 m and thickness of 0.11 m. An initial impact velocity V_0 is prescribed to the impacting mass. Due to the symmetry of the problem a quarter of the foam block is modeled. The velocity V(t) and corresponding displacements u(t) of the rigid mass are recorded.

The crushable foam model with volumetric hardening available in ABAQUS/Explicit (ABAQUS 6.11) was used for the foams. Elastic modulus of 5 GPa is used in the simulation to Latin American Journal of Solids and Structures 12 (2015) 905-924

diminish the influence of the elastic deformations. The stress strain curve for Cymat foam (aluminium based closed cell foam) with density 411.5 kg/m3 (Langdon et al., 2010) is shown in Figure 1.

Solid elements were used to model the foam when an automatic meshing was applied in the (y,z) plane with a maximum element length equal to the element thickness in the x direction. The mesh sensitivity analysis was carried out with respect of the variation of the distances of propagation of the compaction waves, thicknesses of the compacted zones, $h_1(t)$ and $h_2(t)$, longitudinal displacements, $u_1(t)$ and $u_2(t)$, and nominal strains. It was established that the mesh refinement with element thickness of 0.74 mm in the x-direction were used in the present analysis.

4 EXAMPLES AND DISCUSSION

4.1 Comparison between the theoretical approach and numerical simulations

The response of a foam block to a mass impact is analyzed for initial velocities between 50 and 200 m/s. For the selected foam configuration a secondary compaction can occur due to the reflected wave, which develops from the stationary end. During Phase 2, the dependence $\mathcal{E}_2(\bar{x}, V)$ is approximated by the function

$$\varepsilon_2(\overline{x}, V) = \frac{a_2 + b_2 \overline{x} + c_2 V}{1 + d_2 \overline{x} + f_2 V}$$
⁽²²⁾

where $\bar{x} = (u_2 + h_2)$, $\bar{x} \in [0, L_1]$. The values of the coefficients in Eq (22) are given in Table 1 for $V_0 = 100$, 150 and 200 m/s. The strength variation at $t = t_1$, which corresponds to the variable density, $\rho_1(\bar{x})$, attained by the foam at this time instance is approximated by the function

$$\sigma_{1Y}(\bar{x}) = \frac{a_s(\bar{x}/L_1)}{1 + b_s(\bar{x}/L_1)} + c_s.$$
⁽²³⁾

The coefficients of the functions, which approximate $\rho_1(\bar{x})$ and $\sigma_{1Y}(\bar{x})$, are presented in Tables 2 and 3, respectively.

Coefficients	a_2 [Pa]	$b_2 \; \mathrm{[Pa/m]}$	$c_2 \; \mathrm{[Pa\;s/m]}$	$d_2 \; \mathrm{[Pa/m]}$	$f_2 \; \mathrm{[Pa.s/m]}$
$\mathrm{V_0} = 100~\mathrm{m/s}$	-1.290114e-03	7.691426e-01	1.324590e-02	$2.953083e{+}01$	1.657334e-02
$V_0=150~\mathrm{m/s}$	-1.456592e-03	3.674587e-01	9.540665e-03	$3.028801e{+}01$	1.428836e-02
$\mathrm{V_0}=200~\mathrm{m/s}$	-4.267747e-03	3.117275e-01	8.205523e-03	$3.209627e{+}01$	1.372385e-02

Table 1: Coefficients in Eq. (22) ($\mathcal{E}_2(V, \overline{x})$) for different impact velocities.

Coefficients $[kg/m^3]$	$a_{r,0}$	$a_{r,1}$	$a_{r,2}$	$a_{r,3}$	$a_{r,4}$
$\mathrm{V_0}=100~\mathrm{m/s}$	472.550	25.121	826.803	-1542.069	1140.289
$\mathrm{V_0}=200~\mathrm{m/s}$	648.053	-26.917	1657.715	-3188.078	2303.912

Table 2: Coefficients in Eq. (15) ($\rho_1(\bar{x})$) for initial impact velocities of 100 and 200 m/s.

Coefficients	a_s [Pa]	b_s	$\mathcal{C}_{s}\left[ext{Pa} ight]$
$\mathrm{V_0}=100~\mathrm{m/s}$	$9.234171e{+}05$	-8.727554e-01	$1.898423e{+}06$
$\mathrm{V_0}=200~\mathrm{m/s}$	$2.626587 e{+}06$	-8.750500e-01	$3.6508126e{+}06$

Table 3: Coefficients in Eq. (23) ($\sigma_{1Y}(\bar{x})$) for initial impact velocities of 100 and 200 m/s.

Velocity attenuations as functions of the displacement of the rigid mass u are shown in Figure 4 for several impact velocities where the thick lines represent the results from the theoretical approach based on Eqs (6) and (18). The dashed parts of these lines correspond to the velocity attenuation due to the reflected wave as predicted by the theoretical model. The thin lines show the results from the numerical simulations. Good agreement between the two sets of results is observed.



Figure 4: Velocity attenuations as functions of the displacement of the rigid mass; the thick and thin lines represent the theoretical and numerical results, respectively, the dashed lines show the velocity attenuation due to the reflected wave (Eqs. (18)).

The strain variation during the response is the major parameter which, together with the corresponding dynamic stress, determines the energy absorption of the foam and force transfer to the opposite end of the finite thickness block. The strain distributions at the end of the two phases of deformation are shown in Figures 5a - 5d, where the thick lines represent the results of the present theoretical approach and thin lines correspond to the numerical simulations. The present approach to the foam compaction predicts that the kinetic energy resulting from a 50 m/s impact is entirely absorbed within the primary compaction wave when leaving a part of the foam undeformed (Figure 5a). This result agrees favorably with the corresponding numerical predictions.

It appears that the analyzed foam thickness of 0.11 m is not sufficient to absorb the initial kinetic energy entirely within the primary wave when relatively high velocity impact is applied as demonstrated in Figures 5b and 5c for a 100 m/s impact. A strain larger than zero, equal to 0.11, occurs at the stationary end at $t = t_1$. The additional strain caused by the reflected wave is presented by a dashed line in Figure 5c. These strains are added to the strains, which developed within the primary wave, to obtain the final strain distribution shown in this figure.

As expected, a further increase of the impact velocity causes larger strains within the primary wave as well as larger strains within the reflected wave, as shown in Figure 5d for a 200 m/s impact. Moreover, the reflected wave propagates a longer distance (Figure 5d) when comparing it with the case of 100 m/s impact (Figure 5c).



Figure 5 (a): Strain distributions in primary and reflected compaction waves; the thick and thin lines represent the theoretical and numerical results, respectively; (a) Strains within the primary wave, $V_0 = 50$ m/s



Figure 5 (b,c): Strain distributions in primary and reflected compaction waves; the thick and thin Lines represent the theoretical and numerical result, respectively; (b) Strains within the primary wave, $V_0 = 100 \text{ m/s}$; (c) Final strains and strains within the reflected wave, $V_0 = 100 \text{ m/s}$.



Figure 5 (d): Strain distributions in primary and reflected compaction waves; the thick and thin lines represent the theoretical and numerical results, respectively; (d) Strains within the primary and reflected waves together with the final strains, $V_0 = 200 \text{ m/s}$.

2D illustrations of the final strain distributions as obtained from ABAQUS simulations are shown in Figure 6a, b for impact velocities of 50 and 200 m/s, respectively. Non-uniform strains behind the primary compaction wave are well distinguished in both examples. A virtually undisturbed region near to the stationary end is observed for the 50 m/s impact while a region with nearly constant final strains is evident near to the stationary end for the 200 m/s impact.



Latin American Journal of Solids and Structures 12 (2015) 905-924

It can be shown that while the speed of propagation of the primary compaction wave, $G_1(t)$, depends only on the velocity V(t) during the first phase of deformation, while the speed of the secondary compaction wave, $G_2(t)$, depends on the history of the deformation process during Phase 1 (Figures 5c and 5d) as different levels of compaction are achieved at $t = t_1$ depending on the impact velocity. The speeds $G_1(t)$ and $G_2(t)$ of the primary and reflected waves, respectively are shown in Figure 7 for impact $V_0 = 100$ and 200 m/s. The Lagrangian wave speeds are shown by solid lines while the dashed lines represent the Eulerian speeds of the reflected compaction wave. One can see that the reflected wave, which propagates in a denser material, has a higher speed.



Figure 7: Speeds of propagation of primary, $G_1(t)$, and reflected, $G_2(t)$, compaction waves; the thick lines represent the theoretical results according to Eqs. (6c) and (18c) and numerical results; the thin line shows $G_1(t) = V/\varepsilon_D$ due to the RPPL model.

The present approach is also capable of predicting the maximum stress that occurs at each foam cross-section during deformation. A comparison between the theoretical prediction of the maximum stress and the stress variation at a number of cross-sections due to the FE simulations is presented in Figure 8. It is evident that the theoretical model can capture well the magnitude and the propagation time of the primary and reflected wave in the foam.



Figure 8: Maximum stresses along the foam rod within the primary and reflected compaction wave according to the theoretical model and the stress variation with time at a number of cross-sections predicted by the FE model; the thick and thin lines represent the theoretical and numerical results, respectively.

4.2 Comparison Between the Present Theoretical Approach and a Model with a Predefined Densification Strain

The classical RPPL model (Reid and Peng, 1997) is frequently used to predict the velocity attenuation and energy absorption of various cellular materials under impact loading. Therefore this model is used here to distinguish between the dynamic compaction of a foam material, which exhibits strain hardening and its approximation by the RPPL model.

The approximation of the actual material by the RPPL model removes the unique stress-strain dependence of the cellular material and the definition of the model parameters in terms of the critical, σ_{cr} , (or plateau) stress and densification strain, \mathcal{E}_D , becomes crucial. Since the densification strain is not strictly a physical quantity, different definitions for this parameter have been suggested. Obviously, a complete densification approaching the density of the constituent material can be achieved mainly in very low density cellular materials subjected to a high velocity impact or other very high intensity loads. A physically rigorous approach to obtain \mathcal{E}_D was proposed by Tan et al. (2005a) providing more realistic estimates for the parameters of the RPPL model. The latter approach is used herein to obtain the characteristic parameters, $\mathcal{E}_D = 0.405$ and $\sigma_{cr} = 2.49$ MPa for the Cymat material analyzed in the present paper and shown in Figure 1.

The great advantage of the RPPL model is that a closed form solution for the velocity attenuation as obtained according to the expression given by Harrigan et al. (2010), which in the current notations reads

$$V(u) = \left\{ \frac{\varepsilon_D}{\rho_0} \left(\left[\left(V_0^2 \frac{\rho_0}{\varepsilon_D} + \sigma_{cr} \right) m^2 \right] \left(m + u \frac{\rho_0}{\varepsilon_D} \right)^{-2} - \sigma_{cr} \right) \right\}^{1/2}.$$
(24)

Assuming a constant predefined densification strain, the critical thickness of the foam, defined as

$$L_{RPPL}^{CR} = \frac{m}{\rho_0} \left(\sqrt{1 + \frac{\rho_0 V_0^2}{\sigma_{cr} \varepsilon_D}} - 1 \right)$$
(25)

was also obtained by the same authors (Harrigan et al., 2010).

The velocities attenuations predicted by the RPPL model and currently proposed approach are shown in Figure 9 for initial impact velocities of 100 and 200 m/s. The predictions due to the numerical simulations (thin lines) are also added for comparison purposes. In both examples, the stroke u (displacement of the rigid mass) is underestimated by the RPPL model thus overestimating the energy absorption capacity of the foam. Moreover, the predictions of the RPPL model for $V_0 = 200$ m/s are valid only for $u \leq 0.0445$ m as at this displacement of the impacting mass, the compaction wave reaches the stationary end. The latter model has no capability to deal with the reflected wave due to assumption of a locking strain.



Figure 9: Comparison between the velocity attenuation predicted by the current and RPPL material model for $V_0 = 100$ and 200 m/s.

The assumption of the predefined strain \mathcal{E}_D is the major reason causing the disparity between the RPPL model predictions and actual deformation of a foam material when this model is applied to approximate foam with strain hardening (Karagiozova et al., 2012). This assumption leads to an incorrect estimate of the dynamic stress enhancement and in general overestimates this value. The comparisons between the dynamic stresses with the velocity attenuation presented in Figures

10a and 10b for two impact velocities show that a considerable stress overestimation results from the application of the RPPL model.

The underestimation of the maximum stroke is also related to an incorrect evaluation of the minimum foam thickness, which is sufficient to absorb a given impact energy. According to Eq. (25), $L_{RPPL}^{CR} = 0.0578$ m for $V_0 = 100$ m/s and 0.146 for $V_0 = 200$ m/s. In reality, the entire thickness of the 0.11 m foam block undergoes compaction at different strain levels (Figures 5b-5d) for both analyzed velocities whilst the initial impact energy cannot be absorbed by the propagation of the primary compaction wave. A secondary compaction occurs and the stresses, which develop within this wave, are not negligible, particularly for the 200 m/s impact, as shown in Figure 10b.

In order to complete the comparison between the predictions of the RPPL model and the current approach, the compaction wave speed defined by the RPPL model, $G_1 = V/\varepsilon_D$, is added in Figure 7. A considerable overestimation of the wave speed is observed, particularly for the higher particle velocities due to the constant value of ε_D .

It should be emphasized that a definite densification strain related to the material topology or density does not exist in metallic foam materials which exhibit considerable strain hardening (such as Cymat) as the plastic strains are velocity dependent. The compaction strains decrease rapidly with the propagated distance, which is particularly notable in the case of a small mass impact. The foam undergoes secondary compaction in order to absorb the initial kinetic energy. However, all models of foam compaction, which assume a locking mechanism, do not allow the analysis of the reflected wave. Therefore, despite the simplicity of the material models with predefined densification strains, they are not applicable to an impact by a small mass or/and to cellular materials with noticeable strain hardening.



Latin American Journal of Solids and Structures 12 (2015) 905-924



Figure 10 (a, b): Maximum dynamic stresses as functions of the velocity attenuation: current model predictions and predictions of the RPPL model; (a) $V_0 = 100 \text{ m/s}$; (b) $V_0 = 200 \text{ m/s}$.

5 CONCLUSIONS

The unified approach for dynamic compaction of a class of cellular materials exhibiting strain hardening proposed in previous studies (Karagiozova et al., 2010, 2012, 2013, 2014) was extended in this paper in order to apply it to the foam compaction within the wave reflected from a stationary boundary. The latter compaction is considered as a wave of strong discontinuity propagating in a media with non-uniform initial density, which increases monotonically in the direction of loading. The theoretical predictions are compared with FE simulation and good agreement is demonstrated. The proposed approach provides a reliable estimate of the energy absorption capacity of a finite thickness foam block subjected to a uniaxial impact. This approach allows one to obtain the stress and strain variation within the reflected wave which has not been taken into account by the previously proposed models for compaction of cellular materials.

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