

## 1 In-Situ Quasi-Static and Dynamic Behavioural Response of Steel Tubular Frames Subjected to Lateral Impact Loads

### Abstract

Steel tubular members are widely used as primary and secondary structural framing members in offshore oil and gas platforms. A platform is inherently liable to collisions from ships which can create severe structural damages in the rig. The effect of this damage has been studied by a number of researchers through investigating the impact behaviour *isolated* tubular members. This is while, the *in-situ* response of a member located in a structural frame, to lateral impact loads, is not necessarily the same as the response of an individual *isolated* impacted member. In this paper the behaviour of a chord member forming part of a tubular frame, subjected to impact loads, has been investigated. The tubular frame was tested experimentally by other researchers and reported in the literature. The non-linear numerical models of the frame presented by the authors have been validated against the experimental results. These validated models have been examined under both quasi-static and dynamic impact loads with operational pre-loading applied. It has been found that, in a pre-loaded frame, quasi-static impact loading results in the failure of the impacted member. Interestingly, dynamic modelling of the impact results in the dynamic instability of an adjacent bracing member. It has been noticed that, under a dynamic impact, the impacted *in-situ* member (located in the frame) behaves rather similarly to a pin ended *isolated* member. With a quasi-static impact, the impacted *in-situ* member follows fairly closely the response obtained for a fixed ended *isolated* member.

### Keywords

Impact, Frame, Tubular, Dynamic, Quasi-static and Failure

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## 2 NOMENCLATURE

	$\sigma_y$	Material yield stress
	$D$	Tube diameter
	$E$	Material modulus of elasticity
	$F$	Concentrated lateral impact load
	$F_o$	Dynamic lateral step load
	$F_P = 8D^2 t \sigma_y / L$	Plastic collapse load of a tubular beam in pure bending
	$I$	Moment of inertia of the cross-section
3	$L$	Tube length
	$\bar{m}$ $m$	Mass per unit of length
	$P$	Push over load
	$P_y = \pi D t \sigma_y$	Axial squash load of the tube
	$P_u$	Ultimate axial load of the tube
	$R$	Tube radius
	$t$	Tube wall thickness
	$T$	Natural period of vibration

## 4 1 INTRODUCTION

5 Design of offshore structure components against a ship collision is generally based on available  
6 knowledge of the behaviour of damaged and impacted *isolated* tubular members. The majority  
7 of previous studies presented in the literature on ship-offshore collisions concentrate on the  
8 behaviour of *isolated* tubular members (Zeinoddini *et al.* [21, 22]). The behaviour of an  
9 impacted member when it is part of a structural frame is not necessarily the same as the  
10 behaviour of the corresponding *isolated* member, remote from the framework, subjected to a  
11 similar impact load.

12 The differences between the response of a tubular member which is part of a structural  
13 frame (*in-situ* member) and an *isolated* member, when both are subjected to similar impact  
14 loads, are caused by several parameters. Interaction between the global modes of vibration in  
15 the frame and the member modes of deformation of the impacted member can create changes  
16 in the member response. In addition, the inertia forces and damping effects in the structural  
17 frame are different to those of an *isolated* member. The boundary conditions of an impacted  
18 *in-situ* member are a function of the connection properties and the stiffness of other members  
19 meeting at the connection. This type of semi-rigid boundary condition is different from the  
20 typical rigid or free end conditions considered in the literature for the study of *isolated* tubular  
21 members (Yao *et al.*[19]; Frieze and Cho[6]; Amdahl and Eberg[3]; Rambech and Dahl[12]; and  
22 Ricles and Bruin[13]). The boundary conditions of an *in-situ* impacted member are also likely  
23 to change during the impact. This occurs as a consequence of deformations in other members  
24 and in the connections.

25 The main objective of the current study is to investigate the difference between the response  
26 of an impacted member which is part of a structural frame (an *in-situ* member) and the  
27 response of a similar *isolated* member. Such a study can link knowledge available in the

28 literature on the behaviour of impacted and damaged *isolated* tubular members to *in-situ*  
29 response which is closer to real behaviour.

## 30 2 THE BENCH MARK TUBULAR FRAME

### 31 2.1 Test Programme

32 Structural frames have been tested experimentally by other researchers under vertical loads,  
33 lateral loads and base excitations (Martin and Villaverde[9]; and Mosalsm *et al.*,[10]). However,  
34 the authors of the current study are not aware of any relevant impact testing of tubular frames  
35 which can represent ship collisions with offshore structures. Large-scale tubular frames have  
36 also been tested under static push over loading by other researchers to study the failure of  
37 jacket frames (Grenda *et al.*[7]; Nichols *et al.*[11]; and Bolt[4]).

38 The bench mark tubular frames mentioned above were found to be the most relevant  
39 available experimental work that could be used for validating the numerical impact model.  
40 The experimental results from these bench mark tubular frames have been used in this study  
41 for validating the non-linear numerical models of the frame with push over loading. If the  
42 non-linear numerical model can correlate with a push over collapse experiment, this provides  
43 some basis for using the model for the prediction of the dynamic behaviour, although it must  
44 be admitted that the accuracy of the some aspects of dynamic modelling remains uncertain.

45 The experimental data, which were used for benchmarking the non-linear finite element  
46 models of tubular frames, emanate from the Phase I Frames Test Programme carried out in  
47 the placecountry-regionUK. This experimental push over test project was initiated in 1987.  
48 The programme was conducted by Billington Osborne-Moss Engineering Limited (BOMEL)  
49 as part of a joint industry programme with the object of providing test data on the collapse  
50 behaviour of jacket structures and in addition, to develop calibrated software for the non-linear  
51 push over analysis of framed structures (Bolt *et al.*[4]). The Phase I Frames Test Programme  
52 was completed in 1990 and provided the first large-scale test data on the collapse performance  
53 of frames representative of offshore structures. The results of this programme were released  
54 from confidentiality in 1993 (Lalani *et al.*[8]).

55 The Phase I Frames Test Programme consisted of testing four, two bay, X-braced frames.  
56 These tubular frames (*Figure 1*) were the largest frameworks ever to be tested to collapse in  
57 a controlled manner, and provided a new and important insight into the role of redundancy  
58 and particularly tubular joint failures within a frame, neither of which have been investigated  
59 in earlier research programmes (Bolt *et al.*[25]).

### 60 2.2 Numerical Models of the Bench Mark Frame

61 The ABAQUS[1] non-linear finite element program has been used to produce two identical  
62 numerical models of the bench mark frame 1. In the first numerical model, each vertical,  
63 horizontal and diagonal member of the frame shown in *Figure 1* has been modelled using  
64 up to 20 beam elements (type *PIPE31*). The connections are considered to be rigid and the  
65 frame supports are pinned. The second numerical model is the same as the first except that

66 one of the chord members in the upper bay has been modelled using shell elements. Twenty  
67 four shell elements ( $S4R$ ) have been used in the circumference and fifty in the longitudinal  
68 direction of the chord member. The circumferential nodes at the two ends of this member have  
69 been linked to the end node of their adjacent beam element using the multi-point constraint  
70 option available in ABAQUS.

71 The second model allows for local deformations in the chord member which is important  
72 in a ship impact study. The first numerical model excludes local deformation and denting.  
73 Comparing the results obtained from the first and the second models reveals the effect which  
74 local deformations have on the response of the frame.

75 The first model, using only beam elements, is similar to models which have been used in  
76 the design against ship collision for the majority of existing offshore structures. The limited  
77 past capacity of computational facilities did not allow a time consuming analysis capable of  
78 including local effects in the appraisal (Sterndorff *et al.*[16]; and Waegter and Sterndorff[18]).  
79 It should be mentioned that no imperfection has been considered in the above mentioned  
80 numerical models.

### 81 2.3 Validation of Numerical Models

82 *Figure 2* shows the horizontal load-displacement curves from the experimental results and in  
83 addition from the two numerical models when the frame is subjected to a push over horizontal  
84 load at its top. It can be seen that there is a good agreement between the test and the  
85 numerical results. Under push over loading no difference was found in the response of both  
86 of the numerical models. As a result, one curve in *Figure 2* represents the response of both  
87 numerical models, with and without local deformations included.

88 Buckling of the compression brace at the top half of the upper bay was reported to have  
89 caused failure in the test specimen. The same phenomenon was observed to have occurred  
90 in the numerical models. The ultimate lateral capacity of the frame tested was found to be  
91 920kN. The lateral capacity predicted by both the numerical models was found to be 932kN.

## 92 3 RESPONSE TO QUASI-STATIC IMPACT LOADS

### 93 3.1 Models of the Isolated Impacted Tubular Members

94 The two numerical models of the test framework have been examined under lateral impact loads  
95 applied at mid length of a chord member in the frame upper bay. To be able to compare the  
96 results from the *in-situ* tubular members to the response of impacted *isolated* tubular members,  
97 two *isolated* chord members have also been modelled. Again in one model shell elements have  
98 been used which allow for the inclusion of local effects while in the other numerical model beam  
99 elements have been employed which model bowing but do not model local denting deformation  
100 behaviour.

101 As a result, four numerical models have been considered; namely a frame with local effects;  
102 a frame without local effects; an *isolated* chord member with local effects and an *isolated* chord  
103 member without local effects. The geometry and other properties of the *isolated* chords are

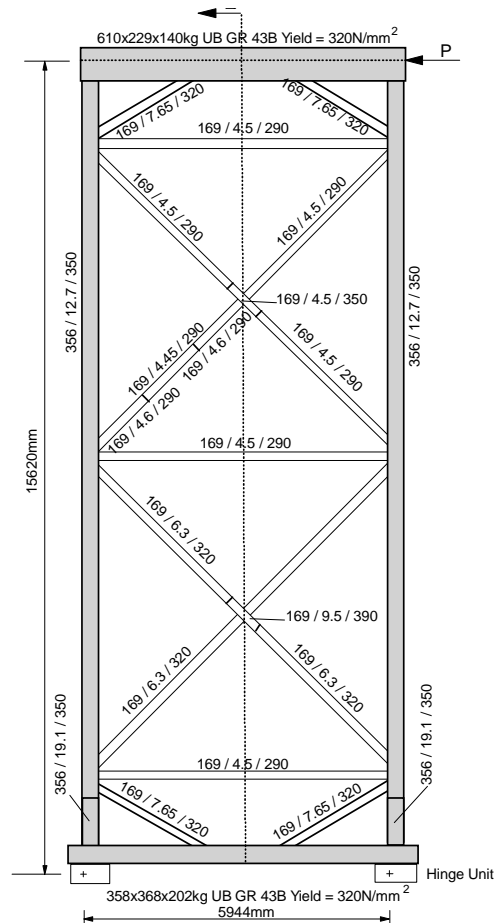


Figure 1 Elevation and properties of the tubular frame (frame 1), used in the benchmarking exercise<sup>1</sup> (Nichols *et al.*, 1994)

104 the same as those of the impacted member in the frames.

105 The boundary conditions at both ends of the *isolated* chord members allow for free axial  
 106 sliding at each end of the member. However, ends are completely restrained against rota-  
 107 tion. This boundary condition models, as closely as possible, the boundary conditions of the  
 108 impacted member in the test frame although some axial restraints will be present.

### 109 3.2 Pre-Loading

110 A ship collision usually occurs when an offshore structure is carrying its operational load. The  
 111 gravity and operational loads (called here pre-loading) can exacerbate the level of structural  
 112 damage caused by a collision. With design safety factors, the operational pre-loading can be  
 113 assumed to be, at maximum, between 50% and 70% of the ultimate axial load of the member  
 114 ( $P_u$ ). The former is the more likely load for a collision event and the latter is mostly associated  
 115 with severe environmental load conditions. In this study only vertical pre-loading in the frame

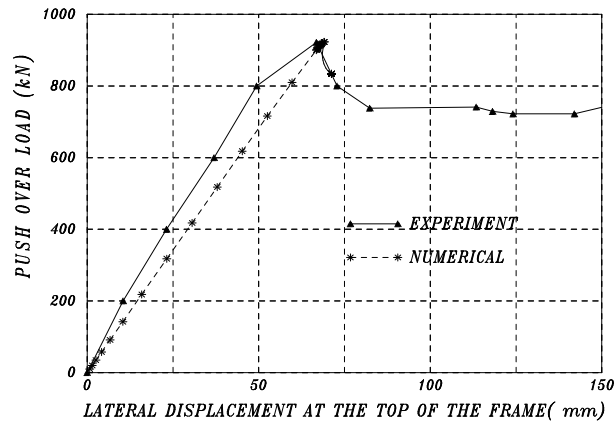


Figure 2 Lateral push over load-displacement curves for test frame 1.

116 members (or axial compression in the *isolated* chord members) has been considered. The pre-  
 117 loading in all models produces an axial compression in the chord member equal to 50% of the  
 118 axial squash load of the member ( $P_y$ ).

### 119 3.3 Response of Pre-Loaded Tubulars to Quasi-Static Impacts

120 The four main numerical models have been studied under a quasi-static lateral impact load  
 121 applied at the mid-span of the chord member. A Modified Riks method of analysis has been  
 122 used because an impacted member experiences local and/or global instabilities in the tube  
 123 wall or in overall. In non-linear finite element analysis, the Modified Riks Method is used for  
 124 unstable static problems such as those found in post-collapse or post-buckling behaviour. The  
 125 tangent stiffness matrix can be examined at any stage of the loading to determine the existence  
 126 of negative eigenvalues. These can then be used to define the type of instability occurring in  
 127 the structure. *Figure 3* shows the response of the four main numerical models to a quasi-static  
 128 impact load. The ordinate represents the dimensionless lateral load. The abscissa in *Figure 3*  
 129 shows the dimensionless lateral deformation at the position of the impact load.

130 When local effects are included in the numerical model, the maximum lateral load which  
 131 can be resisted by the *in-situ* member is about 40% less than the maximum load carried when  
 132 local effects are ignored. The ratio of corresponding values for the *isolated* chord member is  
 133 about 33% with the model including local effects producing the lower value (*Figure 3*). The  
 134 figure underlines the importance of including local effects in a collision study.

135 Both *isolated* and *in-situ* chord members support similar maximum lateral loads when the  
 136 local effects are excluded. The similar values for the maximum lateral loads obtained for  
 137 the *isolated* and *in-situ* members indicate that with quasi-static impacts the rigid boundary  
 138 conditions, considered for the *isolated* chord member, are fairly close to the real circumstances  
 139 found in the structural frame.

140 In the numerical models where local effects have been included, the peak load in the *isolated*  
 141 member is slightly higher than the peak load obtained for the *in-situ* member. Reduction in

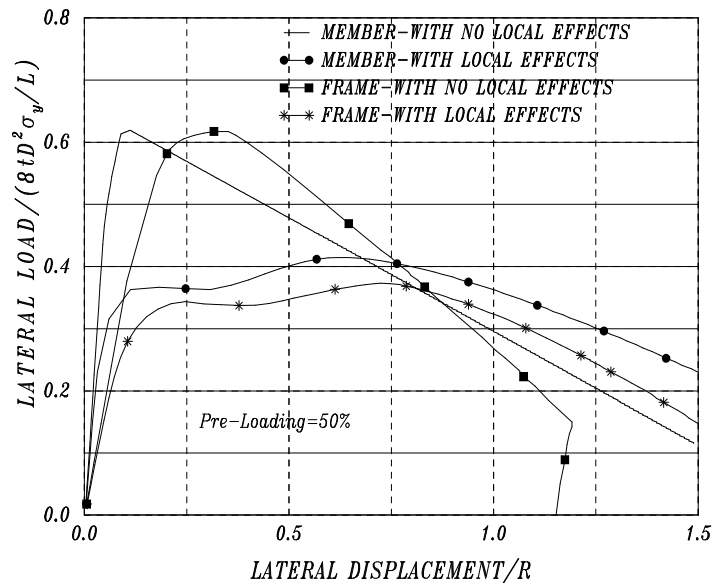


Figure 3 Lateral load-displacement behaviour of the *isolated* and *in-situ* tubular chord members subjected to lateral quasi-static impact.

142 the lateral load capacity indicates that the two ends of the damaged *in-situ* chord member  
 143 exhibit a semi-rigid behaviour compared with the fully restrained condition for the *isolated*  
 144 member. It will be seen later that with dynamic impact loads, considerably higher levels of  
 145 semi-rigidity appear at the ends of the *in-situ* members.

#### 146 4 RESPONSE TO DYNAMIC IMPACT LOADS

147 The four numerical models of frames and *isolated* members outlined in Section 3.1 have been,  
 148 this time, examined under dynamic impact loads. As for previous models a pre-loading has  
 149 been applied with all models and then lateral impact has been applied. An implicit incremental  
 150 direct integration approach, based on the 'Newmark method', has been applied using the  
 151 finite element program ABAQUS. In order to acquire a pure response which is not affected  
 152 by modification of external loads and inertia forces during the impact, step lateral loads have  
 153 been used initially hence providing a better understanding of the dynamic characteristics of  
 154 the impact.

155 In the current study no structural damping has been incorporated into the finite element  
 156 models. The only existing damping in the analysis is the numerical Hilber-Huge damping  
 157 incorporated in the finite element program. This damping is relatively small compared to  
 158 structural damping. It should be mentioned that

159 Obviously, some structural damping forces are involved in the response of the tubes to a  
 160 dynamic load, but damping forces are generally believed not to impose a significant influence  
 161 during the impact. This is because the duration of the impact is usually very short. In ordinary

162 structures, impact forces need a duration several times greater than the fundamental natural  
 163 period of the system to have a remarkable effect on the response (Zeinoddini *et al.*[25]).

#### 164 4.1 Stable Responses

165 *Figure 4* shows the response of the four main numerical models when a dynamic step lateral  
 166 load of  $160\text{kN}$  ( $=0.22[8tD^2\sigma_y/L]$ ) is applied at the mid-span of the chord member. Time  
 167 histories of the deformation at the mid-span of the impacted chord member are displayed in  
 168 this figure. All the models carry a pre-loading which produces an axial compressive force equal  
 169 to 50% of the squash load in the chord member.

170 It can be seen in *Figure 4* that with 50% pre-loading and the 160 kN lateral step load,  
 171 all the responses remain bounded and therefore stable. Models with local effects show larger  
 172 displacements, essentially due to the denting of the chord wall. Some small fluctuations can  
 173 be observed in the response of the models where local deformations have been included. The  
 174 responses of the models where local deformations have been omitted are smooth and free of  
 175 these small fluctuations. The fluctuations appear to be produced by the excitation of higher  
 176 modes of vibration within the tube wall.

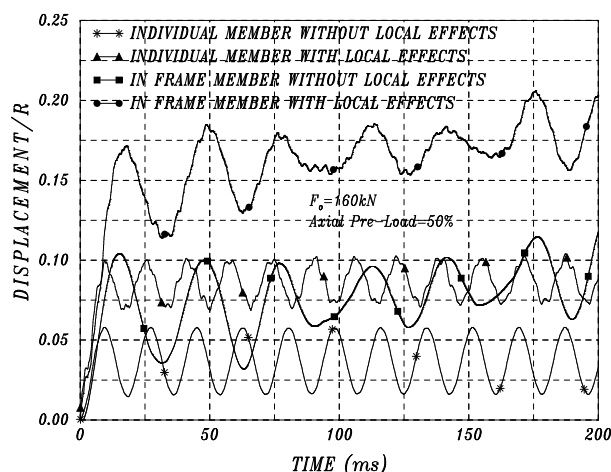


Figure 4 Time history of lateral deformation of the tubular frames and *isolated* members subjected to a dynamic lateral impact.

177 Despite the difference in the level of the displacement, and the appearance of the small  
 178 fluctuations, the responses for the two *isolated* members (with and without local effects) have  
 179 similar shapes and frequencies (*Figure 4*). This indicates that the primary vibration in both  
 180 is governed by the bowing mode of vibration in the member. The oscillations of the structural  
 181 system take place about a displaced configuration. This similarity also exists between the  
 182 responses of the two *in-situ* members.

183 It can be deduced from *Figure 4* that the predominate frequencies, the bowing frequencies,  
 184 in the response of the *in-situ* members are lower than the governing frequencies from the  
 185 corresponding *isolated* member models. Differences between the end conditions of the chord



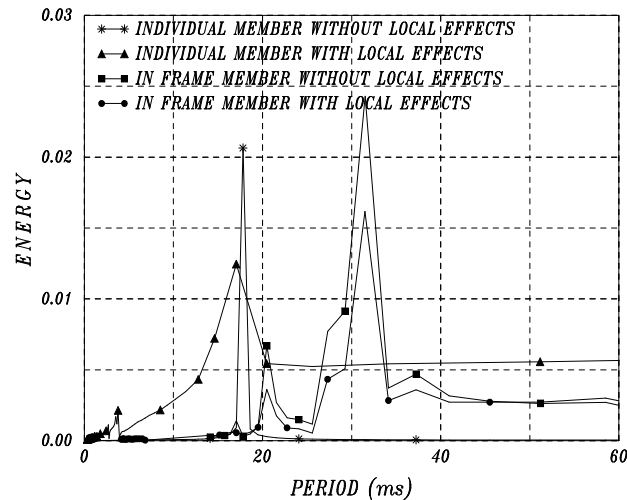


Figure 5 Period extraction of the responses displayed in *Figure 4* using Fourier transformation.

186 member in the frame compared with those used for the *isolated* member models seem to be the  
 187 main source of the difference between the predominate frequencies. As mentioned in Section  
 188 3.1, the end conditions for the *isolated* member models are fixed to prevent rotation but allow  
 189 translation along the axis of the member.

190 *Figure 5* shows the periods of the responses of *Figure 4*, extracted using a Fourier transfor-  
 191 mation. The ordinate shows the energy spectra corresponding to each period and the abscissa  
 192 shows the periods. The periods associated with the peak energies show the predominant peri-  
 193 ods of the vibrations for a stable response presented in *Figure 4*.

194 The dominant periods of the main frame models as well as the *isolated* members, with and  
 195 without local effects included, are given in *Table 1*. The values shown in this table are obtained  
 196 from *Figure 5*. The theoretical first natural periods of vibration for *isolated* tubular members  
 197 with pinned and rigid ends are also given in *Table 1*. The Equations used for the calculation  
 198 of the theoretical first natural period for vibration of pinned and encastred *isolated* tubular  
 199 members are respectively (Clough and Penzien[5]) :

$$T = \frac{2}{\pi} \sqrt{\frac{\bar{m}L^4}{EI}} \quad (1)$$

$$T = \frac{8}{9\pi} \sqrt{\frac{\bar{m}L^4}{EI}} \quad (2)$$

200 It can be seen from *Table 1*, *Figure 4* and *Figure 5* that the dominant period of vibration  
 201 of *isolated* members with and without local effects are close to the theoretical natural period  
 202 of the corresponding tubular member with rigid ends. This shows the agreement between the  
 203 numerical and the theoretical results.

Table 1 The dominant periods of vibration of the four main numerical models within a stable response.

Description	Period of vibration (ms)
<i>Isolated</i> member with no local effects	17.80
<i>Isolated</i> member with local effects	17.05
<i>In-situ</i> member with no local effects	31.50
<i>In-situ</i> member with local effects	31.50
Rigid end tubular member (theoretical)	16.74
Pinned end tubular member (theoretical)	37.96

204 With *in-situ* members, the predominant periods of vibration are almost twice those for the  
 205 *isolated* members. The *in-situ* periods are much closer to the theoretical period of a member  
 206 with pinned ends. This shows that under a dynamic impact, the end conditions for the chord  
 207 member in the frame behave like pin connections. In Section 3.3 it was noted that for a  
 208 quasi-static impact the end conditions for the chord member in the frame followed closely the  
 209 response obtained from rigid end conditions.

210 Table 1, Figure 4 and Figure 5 show that the periods of an *isolated* member model, where  
 211 local effects have been excluded, are less than the model where local effects have been consid-  
 212 ered (the frequencies are higher). This unexpected stiffer behaviour could be due to the fact  
 213 that the dent produced in the impacted members is not sympathetic to the member bowing  
 214 mode shapes and therefore the dented members become slightly stiffer for vibrations in the  
 215 bowing mode. With the *in-situ* member, local damage again produces a slightly higher stiff-  
 216 ness in the impacted member compared to that for the models where local effects have been  
 217 ignored.

## 218 4.2 Unstable Responses

219 The oscillations presented in Figure 4 are bounded and therefore the structural systems remain  
 220 stable. By increasing the lateral impact load (or the pre-loading), the response of the structural  
 221 system starts to become unbounded. In each of the four main numerical models, beyond a  
 222 defined level of impact load, the responses become unbounded. This indicates that at this  
 223 load level a dynamic instability has been propagated in the structural system. This dynamic  
 224 limit point load, or the load which results in instability of the structure, is different in each  
 225 model. For a MDOF (Multi Degree Of Freedom) system, an exact solution for the dynamic  
 226 limit point load does not exist. Only Minimum Guaranteed Critical Loads (MGCL) can be  
 227 evaluated (Simitises[14], Zeinoddini et al.[20, 27], Zeinoddini and Parke[25]).

228 Figures 6 and 7 show the response of two of the four main numerical models when different  
 229 dynamic step lateral loads are applied at the mid-span of the chord member. Time histories  
 230 of lateral deformation of the impacted position at mid-span of the damaged chord member  
 231 are displayed in these figures. All the models carry a pre-loading which produces an axial  
 232 compressive force equal to 50% of the chord member squash load ( $P_y$ ). The stable and unstable

233 responses can be distinguished in each figure. It can be seen that beyond a certain level of  
 234 impact load, the responses become unbounded. This indicates that at this load a dynamic  
 235 instability has been propagated in the structural system. This dynamic limit point load or,  
 236 the load which results in instability of the structure, is different in each of the four models.

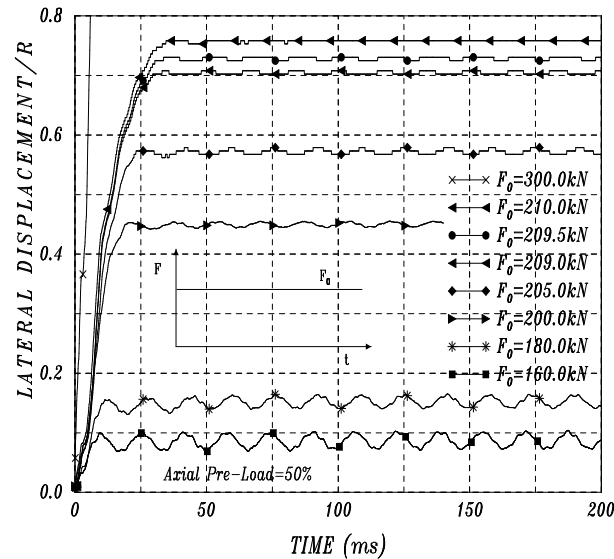


Figure 6 Time history of lateral deformation of the tubular member subjected to a dynamic lateral impact, where local effects have been included.

## 237 5 COMPARING THE QUASI-STATIC AND DYNAMIC RESPONSES

238 The quasi-static and dynamic behaviour of *isolated* impacted tubes have been addressed by  
 239 Zeinoddini *et al.* [23, 24, 26] and Al-Thairy and Wang [2]. In this Section some differences  
 240 observed between quasi-static and dynamic responses of *in-situ* impacted tubular members are  
 241 discussed.

### 242 5.1 Failure Loads and Displacements

243 In *Tables 2 to 4*, the quasi-static and dynamic deformations, failure loads and failure displacements  
 244 of the four basic numerical models used in the current study are compared with each  
 245 other.

246 *Table 2* compares the quasi-static and maximum dynamic lateral displacements under  
 247 an impact load of 160 kN (or  $0.22 (8tD^2\sigma_y/L)$ ) which produces stable responses in all four  
 248 models. The dynamic responses of the four models to this level of impact load have been  
 249 shown in *Figure 4*. The displacements given in *Table 2* are dimensionless lateral deformations  
 250 of the front elevation of the damaged chord member at the position of the impact load. For  
 251 instance, in a frame model with local effects included the lateral displacement given in *Table*  
 252 *2* consists of the summation of denting, bowing and global deformations at the position of the

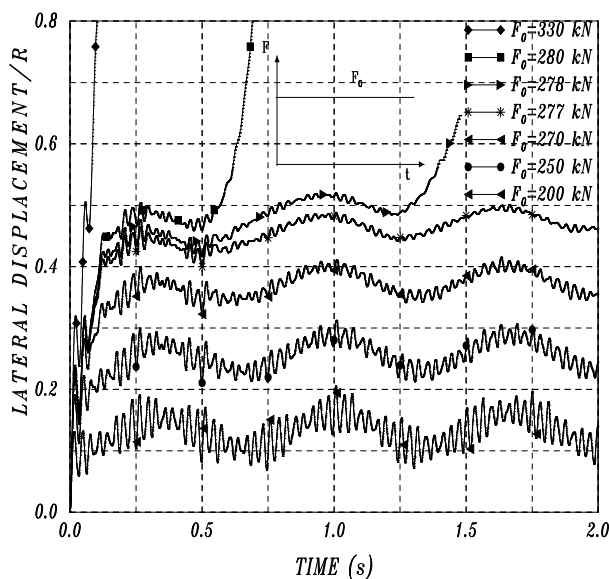


Figure 7 Time history of lateral deformation of the *in-situ* tubular member subjected to a dynamic lateral impact, where local effects have not been included.

253 impact load.

Table 2 Quasi-static and maximum dynamic lateral displacements of the four numerical models under stable impact loading ( $F_0=0.22 (8tD^2\sigma_y/L)$  or 160kN).

Description	Static displacement /R	Dynamic (Max) displacement /R	Dynamic Amplification Factor (DAF)
<i>Isolated</i> member with local effects omitted	0.0229 (1.00)	0.0579 (2.73)	2.53 (1.00)
<i>Isolated</i> member with local effects included	0.0298 (1.30)	0.1039 (4.54)	3.49 (1.38)
<i>In-situ</i> member with local effects omitted	0.0590 (2.58)	0.1180 (5.15)	2.00 (0.79)
<i>In-situ</i> member with local effects included	0.0705 (3.08)	0.2062 (9.00)	2.92 (1.15)

254 To ease comparisons, the displacements have been normalised using the quasi-static re-  
 255 sponse of the *isolated* member with local effects excluded, the normalised figures being given  
 256 in brackets. It can be seen from *Table 2* that the Dynamic Amplification Factor (the ratio  
 257 between the maximum dynamic deformation and the corresponding static deformation) in  
 258 the frame models is smaller than the similar values for the *isolated* member models. This is  
 259 because the frame models have a relatively higher stiffness than the chord member. The Dy-  
 260 namic Amplification Factor has an inverse relationship to the stiffness of the system. Dynamic  
 261 Amplification Factors are higher when local effects are included. This difference is due to the

262 lower stiffness of the tube wall against lateral loading compared to the bowing stiffness of the  
263 member.

264 *Tables 3 and 4* give the quasi-static and dynamic failure loads and failure displacements of  
265 the four main numerical models, respectively. The failure loads are the maximum lateral loads  
266 that can be resisted by the models. The failure displacements are the corresponding lateral  
267 displacements at the mid-span of the impacted member.

Table 3 Quasi-static and maximum dynamic lateral failure displacements of the four numerical models under extreme impact loading.

Description	Quasi-static failure displacement/R	Dynamic failure displacement/R	Dynamic failure displacement/quasi-static displacement
<i>Isolated</i> member with local effects omitted	0.112 (1.00)	0.344 (3.07)	3.07 (1.00)
<i>Isolated</i> member with local effects included	0.618 (5.52)	0.758 (6.77)	1.23 (0.40)
<i>In-situ</i> member with local effects omitted	0.316 (2.82)	0.443 (3.96)	1.40 (0.46)
<i>In-situ</i> member with local effects included	0.725 (6.47)	0.871 (7.78)	1.20 (0.39)

Table 4 Quasi-static and maximum dynamic lateral failure loads of the four numerical models under extreme impact loading.

Description	Static failure load( $8tD^2\sigma_y/L$ )	Dynamic failure load( $8tD^2\sigma_y/L$ )	Dynamic failure load/quasi-static failure load
<i>Isolated</i> member with local effects omitted	0.619 (1.00)	0.393 (0.63)	0.63 (1.00)
<i>Isolated</i> member with local effects included	0.415 (0.67)	0.284 (0.46)	0.68 (1.08)
<i>In-situ</i> member with local effects omitted	0.617 (1.00)	0.385 (0.62)	0.62 (0.98)
<i>In-situ</i> member with local effects included	0.373 (0.60)	0.259 (0.42)	0.69 (1.09)

268 With a quasi-static impact, a maximum load value can be obtained and the corresponding  
269 displacement can be calculated (see *Figure 3*). With a dynamic impact load, no exact failure  
270 load can be defined. The dynamic failure loads given in *Table 4* correspond to the oscillation  
271 immediately below the Minimum Guaranteed Critical Loads. These load values appear to be  
272 close to the exact dynamic failure load. The dynamic failure displacements presented in *Table*  
273 *3* correspond to the load values mentioned above. These may not be close to collapse values  
274 because of the potential rapid change in this critical region.

275 **5.2 Circumferential and Longitudinal Deformations**

276 *Figure 8* compares the dynamic circumferential deformations of the *in-situ* impacted member,  
 277 where local effects are included, with the corresponding quasi-static deformations. The de-  
 278 formed shapes in *Figure 8* show the cross-sections of the damaged tube at its mid-span. It can  
 279 be seen that, with similar levels of denting, no significant difference can be observed between  
 280 the shapes of dynamic and static circumferential deformations. Some researchers believe that  
 281 the dynamic lateral loading of tubular members creates greater local deformations in the tube  
 282 compared with a static lateral load (Søreide and Kavlie[15]; and Stronge[17]). From *Figure 8*  
 283 it does not seem that the dynamic effects themselves have produced more local deformations  
 284 in the circumferential direction.

285 The dynamic impacts did not show any significant differences to the deformations compared  
 286 to the quasi-static impacts. This is because only small values of inertia forces are acting on  
 287 the tube wall. The mass of the tube wall is insignificant compared to the mass of the frame  
 288 and especially to the top-side mass. Consequently, the effect of the inertia forces in the tube  
 289 wall, which can cause different shapes of cross-section in dynamic loading, becomes negligible.  
 290 Therefore the dynamic effects themselves cannot cause greater local effects than the quasi-  
 291 static impacts. A collision produced by an external mass can produce much larger inertia  
 292 forces adjacent to the tube wall of the impacted member. In this case greater differences  
 293 would be expected between quasi-static and dynamic deformations.

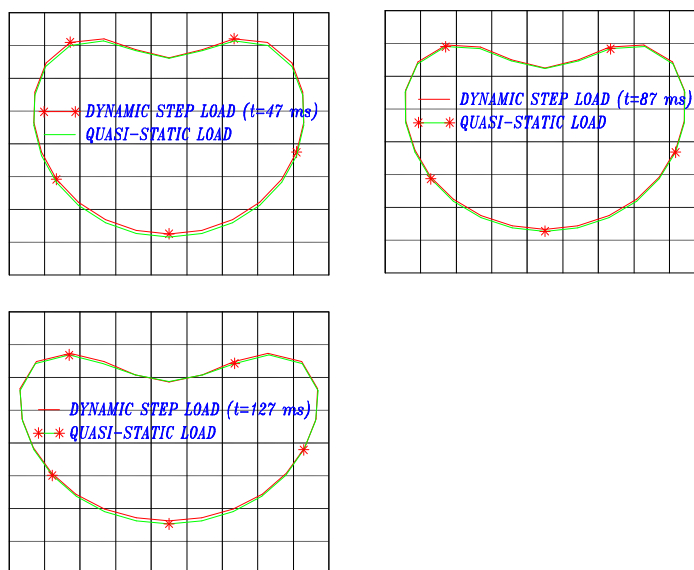


Figure 8 Deformed cross-sections of the *in-situ* chord member subjected to both quasi-static and dynamic lateral impacts.

294 Longitudinal deformations created by corresponding quasi-static and dynamic impact loads  
 295 are compared in *Figure 9*. This figure shows the deformations of the front elevation of the  
 296 tube along the impacted chord. In the abscissa the unity value represents the top of the

297 chord member. Prior to a member buckling in the frame, the corresponding quasi-static and  
 298 dynamic curves shown in *Figure 9* are close to each other with the dynamic deformations  
 299 being very slightly higher. Again dynamic impacts have not caused more local deformation  
 300 compared with the corresponding quasi-static impact.

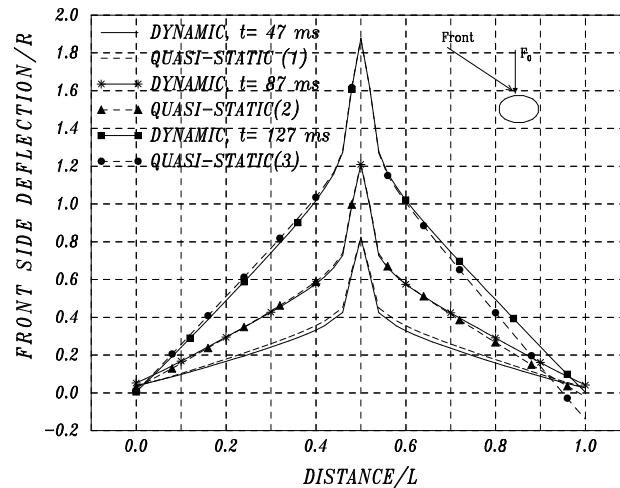


Figure 9 Deformation in the front elevation of the *in-situ* chord member subjected to quasi-static and dynamic lateral impacts.

### 301 5.3 Failure Modes

302 In *Figure 9* after member buckling occurring at  $t=106\text{ms}$  (between curves corresponding to  
 303  $t=87$  and  $127\text{ms}$ ) there is a difference between quasi-static and dynamic deformations in the  
 304 upper part of the impacted member. With a dynamic impact load, the top surface of the  
 305 impacted member moves in the impact direction, while in contrast, under a quasi-static impact,  
 306 the top surface of the impacted member moves in the opposite direction to that of the impact  
 307 load. The displacement actually becomes negative for the quasi-static impact.

308 This difference reflects the occurrence of different failure modes in the tubular frame under  
 309 these two types of impact loads. Under a dynamic impact load, failure occurs first in the  
 310 upper bay compressive brace of the frame (see *Figure 10*). After buckling of this brace (with  
 311 a reduction in the lateral resistance of the frame) there is an increase in the movement of the  
 312 top end of the impacted member in the direction of the impact load.

313 Under quasi-static loading, in contrast to the dynamic impact, failure first takes place in  
 314 the impacted member rather than in the bracing member. After failure, the member shortens  
 315 and the vertical pre-loading produces a rotation of the top of the frame. As a result, the  
 316 impacted member moves in the opposite direction to that of the impact load.

317 Most previous studies on ship-offshore collisions have concentrated on damage caused in  
 318 the impacted member. The current study shows that the consequences of a collision may  
 319 not be limited only to the impacted member and may extend to the connections or adjacent  
 320 members.

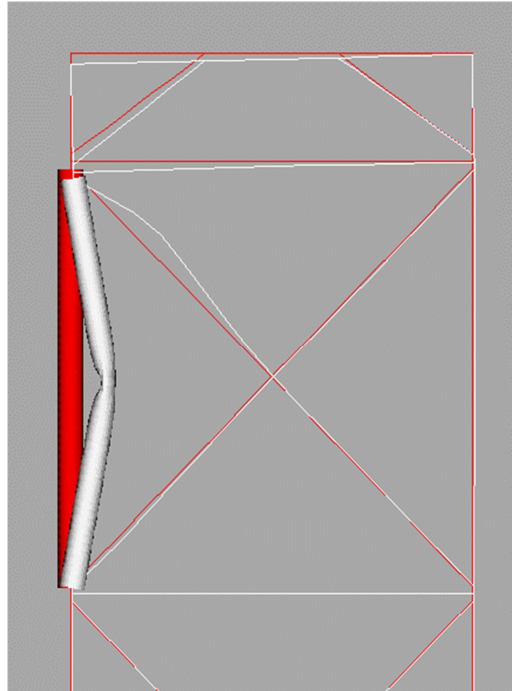


Figure 10 Failure modes in the tubular frame, subjected to a dynamic lateral impact (a brace buckles first).

321 In both frame models studied, with and without local deformations included, first failure  
322 was observed to occur in the impacted member if the impact was quasi-static. Failure occurred  
323 in a brace if the impact was dynamic. Dynamic amplification of the response is thought to  
324 have caused this important difference. When a stable structural system is dynamically loaded,  
325 the system oscillates about a deformed position that can be produced by the corresponding  
326 equivalent static loads. The maximum dynamic deformation will be bigger than the maximum  
327 static deformation achieved with the equivalent static load. With dynamic impacts, the global  
328 lateral displacement of the frame was amplified by the dynamic effects and exceeded a critical  
329 limit of lateral displacement. At this time the deformations in the impacted member were still  
330 far from the critical deformations the member could tolerate. This resulted in a failure in the  
331 bracing member prior to the failure in the impacted member.

332 *Figures 10 and 11* show the deformed shape of the tubular frame, subjected to lateral  
333 dynamic and lateral quasi-static impacted loads. The deformed shapes have similar lateral  
334 displacements at the position of the lateral load. These figures show the structure after the  
335 occurrence of first member buckling. The onset of member buckling in a compressive brace  
336 in the upper bay can be clearly seen in the deformed shape of the frame subjected to the  
337 dynamic lateral impact (*Figure 10*). With quasi-static loading there is more axial shortening  
338 in the impacted member although this difference is not clearly visible in the figure.



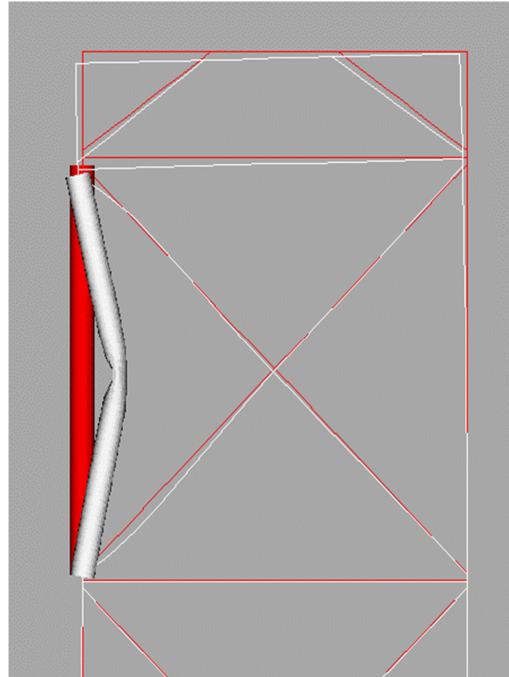


Figure 11 Failure modes in the tubular frame, subjected to a quasi-static lateral impact (the chord member buckles first).

## 339 6 CONCLUDING REMARKS

340 The response of an *in-situ* member in a structural frame subject to a lateral impact load  
341 is not necessarily the same as the response of an *isolated* impacted member. In this paper  
342 the behaviour of a chord member forming part of a tubular frame subjected to impact loads  
343 has been investigated. The tubular frame was tested experimentally by other researchers and  
344 reported in the literature. Non-linear numerical models of the frame have been validated  
345 using the experimental test results. These validated models have been examined under both  
346 quasi-static and dynamic impact loads with operational pre-loading applied. Some of the main  
347 differences between the response of a laterally impacted *in-situ* member and a corresponding  
348 *isolated* member have been reported in this paper.

349 With dynamic loading the impacted *in-situ* member has been found to behave rather  
350 similarly to that expected from an *isolated* member with pinned end conditions. With a quasi-  
351 static impact, the impacted in-situ member follows fairly closely the response obtained for a  
352 fixed ended isolated member. The end rigidity of the tubular member has a direct influence  
353 on the effective length of the member, which can subsequently affect the axial design load  
354 carrying capacity.

355 In the current study no significant difference has been found between the dynamic and  
356 quasi-static circumferential and longitudinal deformations, although some researchers believe  
357 that dynamic impact loads produce more local deformations in tubular members compared

358 with a quasi-static load.

359 Whether or not local deformations have been included, first failure in the frame was ob-  
360 served to occur in the impacted member, if the impact was quasi-static. Failure occurred  
361 in a brace when the impact was dynamic. Most previous studies of ship-offshore collisions  
362 have concentrated on damage caused to the impacted member. The current study shows that  
363 the consequences of a collision may not be limited only to the impacted member but can ex-  
364 tend to the connections and/or adjacent members. Some potential modes of failure might be  
365 overlooked if a quasi-static method of analysis is selected.

366 It should be emphasised that the results presented in this paper do not give a complete  
367 picture of the *in-situ* behaviour of laterally impacted steel tubular members. They are, however,  
368 able to illustrate some of the key factors involved. The tubular frame used in this study was a  
369 two bay, two dimensional frame. Real structures are more complex than this two dimensional  
370 benchmark frame. However, a number of important phenomena have been identified which  
371 will assist in future collapse analyses of complex offshore structures using non-linear software  
372 to determine the ultimate, and residual strengths of platforms.

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